



Vehicle Power Source Supporting Structure

FIELD OF THE INVENTION

5 The present invention relates to a supporting structure of a vehicle power source used for a vehicle body support of a vehicle power source through a plurality of mount members with elasticity.

BACKGROUND OF THE INVENTION

10 Conventionally, ~~as this kind of~~ there is a supporting structure ~~[[of]]~~ for a vehicle power source used in a vehicle body. ~~there are so-called~~ Such supporting structure types include a gravity center mount type of supporting structure and an inertia bearing main shaft type of supporting structure.

15 The gravity center mount type is a type in which a mount center of elasticity center is created through mounts, and is substantially accorded with a gravity center ~~[[and]]~~ of the supporting structure ~~in which~~. In this configuration, an engine and transmission are visually placed and held on the mounts, which are provided on frame members such as ~~[[the]]~~ a sub-frame.

20 When the engine is lengthwise disposed, the center of elasticity center in total created by the mounts is set higher than the gravity center of the engine in many cases by tilting mounts disposed at right and left near the gravity center.

25 ~~Then, as the~~ An inertia bearing main shaft type, such a type as includes a plurality of mount members consisting of a front stopper disposed at a front side of an engine, a rear stopper disposed at a back side of the engine, a side engine mount disposed at a side end of the engine, and a transmission mount disposed at a side end of the transmission ~~is disclosed~~ (for example, refer to

Japan patent publication 63-55453).

Here, the front and rear stoppers are secondary weight sharing mount members which do not mainly share the weight of the engine and transmission being a power source.

5 On the other hand, the side engine mount and transmission mount are primary weight sharing mount members which mainly share the weight of the engine and transmission.

FIG. 1 is a rear view showing a general configuration of a conventional inertia bearing main shaft type of ~~[[a]]~~ supporting structure. As shown in
10 FIG.1, although the inertia bearing main shaft type is generally a type of supporting structure which holds the vicinity of a torque roll shaft, mount members 101 are in many cases disposed on a side frame or at its side in the case of a layout in which an engine 102 is laterally disposed. A position of a gravity center 121 in a power source 104 which is a connected body of the
15 engine 102 and a transmission 103 is mostly located under the side frame. Therefore, positions of a side engine mount 112a and transmission mount 112b at both sides of the power source 104 are located above the gravity center of the power ~~source~~104 source 104 in many cases. Accordingly, ~~[[an]]~~ a center of
elasticity ~~center of total~~ created by the mounts is higher than the gravity
20 center 121 in many cases.

In the gravity center mount type of supporting structure, however, the supporting structure is a type in which the engine and transmission are placed and held on mounts provided on frame members such as the sub-frame, so that
the center of elasticity ~~center~~ formed by the ~~total~~ mounts easily becomes lower
25 than the gravity center consisting of the engine and transmission. Therefore, the power source easily rolls and/or pitches according to movements of a

vehicle.

Moreover, there is a problem that a passenger easily ~~perecepts~~
experiences a roll/pitch feeling of the power source and cannot get a sense of
~~oneness of~~ unity between the power source and vehicle body and so a sufficient
5 drive-safety/ride-quality feeling cannot be obtained.

On the other hand, in a conventional inertia bearing main shaft type of
supporting structure shown in FIG. 1, the mount members 101 are disposed on
the side frame or at its side, ~~so~~ . In this configuration, it is difficult to adopt a
double vibration isolation structure such that mount members 101 are placed
10 and held on the side frame and elastically supported by a vehicle body frame,
~~for the mount members 101~~ compared to the gravity center mount type of
supporting structure. Moreover, there is a problem that routes from the side
engine mount 112a and transmission mount 112b at both sides to a cabin are
short, so a sound and vibration which cannot be completely shut off by the
15 mount members 101 are easily transmitted to a passenger.

SUMMARY OF THE INVENTION

The main problem of the present invention is to provide the supporting
structure of a vehicle power source in which a vibration feeling in the
20 front/rear direction (hereinafter referred to as longitudinal direction) as well as
in the right/left direction (hereinafter referred to as lateral direction) of a
vehicle body can be eliminated so as to surely and sufficiently obtain a
drive-safety/ride-quality feeling.

The supporting structure related to the invention is used for the support
25 to the vehicle body of the vehicle power source through a plurality of mount
members ~~with~~ having elasticity.

The plurality of mount members consist of the primary weight sharing mount members and secondary weight sharing mount members.

Here, at least one of the primary weight sharing mount members is attached at a lower position than ~~[[the]]~~ a height of a gravity center of a power source to ~~[[the]]~~ a sub-frame that is elastically supported by a vehicle body and carries out a function mainly sharing the weight of the power source.

On the other hand, the secondary weight sharing mount members have at least one directional spring component of longitudinal and lateral directions of the vehicle body, are attached to the vehicle body at a higher position than the height of the gravity center of the power source and do not mainly share the weight of the power source.

Furthermore, ~~[[the]]~~ a height of ~~[[an]]~~ a center of elasticity center as a whole and created by the position of the mount members is set at a higher position than that of the gravity center of the power source.

Because the height of the center of elasticity, center of created by the plurality of mount members consisting of the primary and secondary weight sharing mount members as a whole, is set at a higher position than the height of the gravity center of the power source, ~~such~~ the supporting structure enables vibration feelings in the longitudinal direction as well as in the lateral direction of the vehicle body to be eliminated.

Accordingly, the supporting structure enables a vibration isolation effect and a drive-safety/ride-quality feeling with a sense of ~~oneness of~~ unity between the power source and vehicle body to be surely and sufficiently obtained.

In the secondary weight sharing mount members, the spring component elasticity of an upward /downward (vertical) direction is set to be softer than that of the longitudinal or lateral direction.

According to such [[the]] a supporting structure, in the secondary weight sharing mount members elastically supporting both sides of the power source, the spring component elasticity of the vertical direction is set softer (a lower spring constant) and that of the longitudinal or lateral direction is set more rigid (a higher spring constant). Thus, the secondary weight sharing mount members ~~enable a passenger to be~~ make it difficult for a passenger to experience to percept a vibration feeling ~~by curbing, where~~ the roll/pitch of a driving source and a vibration transmission from body side frames can be curbed.

The sub-frame is supported under the body side frames through floating sub-frame support mounts with bolts and buffer members.

According to such [[the]] supporting structure, the sub-frame is elastically supported to the body side frames by the floating sub-frame support mounts. Thus, the driving source, including the engine and transmission, is double-elastically supported (double vibration isolation) by the primary weight sharing mounts and floating sub-frame support mounts.

Moreover, because the sub-frame is supported under the body side frames through the floating sub-frame support mounts, it breaks up, falls downward from the body side frames, and separates in the case of a vehicle collision. Therefore, a passenger compartment of a vehicle may go over the power source ~~goes on the driving source~~ in the case of the vehicle collision, so it can be prevented from being crushed by the driving source.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a rear view showing a general configuration of supporting structure of a vehicle power source in a conventional embodiment example.

FIG. 2A is a plan view of a supporting structure showing a general configuration of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

5 FIG. 2B is a perspective view showing a general configuration of supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

FIG. 3A is a plan view showing a schematic configuration of supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

10 FIG. 3B is a rear view showing a schematic configuration of supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

FIG. 3C is a side view showing a schematic configuration of supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

15 FIG. 4A is a cross-section view showing a supporting structure of a ~~floating sub-frame support~~ mount at the front and back sides of a sub-frame in one ~~of embodiments~~ embodiment of the invention.

FIG. 4B is a cross-section view showing a supporting structure of another embodiment of a sub-frame support ~~floating~~ mount in between a sub-frame.

FIG. 5A is a rear view illustrating a performance of a supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the invention.

25 FIG. 5B is a side view illustrating a performance of supporting structure of a vehicle power source related to one ~~of embodiments~~ embodiment of the

invention.

DESCRIPTION OF THE PREFERRED EMBODIMENTS

5 One ~~[[of]]~~ preferred ~~embodiments~~ embodiment of the invention will be described in detail hereinafter with reference to the accompanying drawings.

~~a. Supporting Structure of Vehicle Power Source~~ a. Supporting Structure of Vehicle Power Source

10 FIGS. 2A and 2B are a plan view and perspective view showing a general configuration of the supporting structure related to one of ~~embodiments~~ embodiment of the invention, respectively. FIGS. 3A, 3B, and 3C are a plan view, rear view, and side view showing a schematic configuration of the supporting structure related to one of ~~embodiments~~ embodiment of the invention, respectively.

15 In the embodiment illustrated, the supporting structure is designed to be used for the support of an engine 2 which is the vehicle power source to a vehicle body 21 through a plurality of mount members 1 ~~with~~ having elasticity so that the engine is elastically supported by the vehicle body.

20 Here, the engine 2 is designed so that a transmission 3 is connected to one end of its crank shaft (not shown in any drawing) and the crank shaft is laterally disposed for the vehicle body 21.

Such the plurality of the mount members 1 are designed to consist of primary weight sharing mount ~~members~~ members and secondary weight sharing mount members [[12]].

25 These respective mount members will be described hereinafter in more

detail.

~~b. Primary Weight Sharing Mount Members 11~~ b. Primary Weight Sharing Mount Members

5 The primary weight sharing mount members [[11]] are designed to be attached at lower positions than ~~the~~ a height of a gravity center 31 in the engine 2 to sub-frames 22 elastically supported to the vehicle body 21, and they mainly play a role of sharing the weight of the engine 2.

10 Concretely, these mount members [[11]] are designed to consist of a ~~[[rear]]~~ front mount 11a disposed at a ~~[[front]]~~ rear side of the engine 2 and a front mount 11b disposed at a back side of the engine 2 and a trans-lower mount (not shown in any drawing).

15 Through ~~these~~ the front mount 11a, rear mount 11b, and trans-mount, the engine 2 is designed to be elastically supported at the sub-frame 22, which is supported by body side frames 23 through floating sub-frame support mounts 13 (see FIGS. 4A and 4B).

20 Meanwhile, FIGS. 4A and 4B are cross-section drawings showing a supporting structure provided between the body side frames 23 and sub-frame 22, FIG. 4A is a drawing showing the floating sub-frame support mount 13 disposed at the front and back sides of the sub-frame 22, and FIG. 4B is a drawing showing the floating sub-frame support mount 13 disposed in between the sub-frame 22. In FIG. 4A, the sub-frame 22 is supported by the body side frames 23 through the floating sub-frame support mount 13 having a bolt 13a and buffer rubber 13b. Although the supporting structure is also nearly same in FIG. 4B, a bracket 13c is provided in between the body frame 23 and sub-frame 22 different from FIG. 4A.

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The front mount 11a, rear mount 11b, and trans-lower mount are designed to be disposed on the sub-frame 22 which is provided to support a lower arm 24 of a suspension.

5 The sub-frame 22, as described before, is floatingly supported under the body side frames 23 of the vehicle body 21, and ~~[[the]]~~ a height of ~~[[an]]~~ a center of elasticity center composed of the front mount 11a, rear mount 11b, and trans-lower mount is designed to be set at a lower position than ~~[[the]]~~ a height of the gravity center 31 in the engine 2.

10 In fact, ~~[[thesel]]~~ the front mount 11a, rear mount 11b, and trans-lower mount ~~taking charge of supporting the self-weight~~ primarily support the weight of the engine 2. The primary weight sharing mount members enable a vibration transmission of a vehicle body to be minimized, because ~~[[they]]~~ the mount members are disposed on the sub-frame and are floatingly supported through the ~~floating~~ sub-frame support mounts 13 for the vehicle body 21, and
15 an engine vibration is double-isolated notwithstanding having a relatively high spring.

~~e. Secondary Weight Sharing Mount Members 12~~ c. Secondary Weight Sharing Mount Members

20 The secondary weight sharing mount members ~~[[12]]~~ have at least one of longitudinal directional and lateral directional spring components and are designed to be attached to the vehicle body 21 at a higher position than the height of the gravity center 31 in the engine 2.

25 The secondary weight sharing mount members ~~[[12]]~~ are designed not to mainly share the weight of the engine 2, and are different from the primary weight sharing mount members ~~[[11]]~~.

Concretely, the secondary weight sharing mount members ~~[[12]]~~ are designed to consist of a side engine mount 12a which is disposed at a right side of the engine 2 and at an end opposite to the transmission 3 of the engine 2 and a trans-upper mount 12b which is disposed at a left side of the engine 2 and on the transmission 3 of the engine 2.

That is, through these side engine mount 12a and trans-upper mount 12b, the engine 2 is designed to be supported by the body side frames 23.

The side engine mount 12a and trans-upper mount 12b are designed to be disposed at a higher position ~~[[for]]~~ than the height of the gravity center 31 in the engine 2.

Therefore, ~~[[the]]~~ a height of ~~[[an]]~~ a center of elasticity center composed of the side engine mount 12a and trans-upper mount 12b is designed to be set at a higher position than the height of the gravity center 31 in the engine 2.

As shown in FIG. 5B, these side engine mount 12a and trans-upper mount 12b enable a relatively small spring to be used and a vibration transmission to the vehicle 21 to be minimized because they work (see a black small arrow mark in FIG. 5B) as ~~a stopper~~ stop members which ~~does not take charge of supporting the self-weight~~ do not primarily support the weight of the engine 2 or as a spring for a control mode ~~[[for]]~~ of a vibratory force (see a black bold arrow mark in FIG. 5B) which is input into tires from the longitudinal direction ~~[[in]]~~ such as when passing a rough road.

Then, in the case of a conventional supporting structure of a vehicle power source with only a side engine mount, a vibration in the longitudinal direction results in becoming rather large, in contrast to ~~different from~~ the supporting structure of the vehicle power source in the embodiment illustrated (see a white arrow mark in FIG. 5B).

Moreover, ~~in such~~ with the plurality of mount members 1, the height of ~~[[an]] a center of elasticity center~~ as a whole is designed to be set at a higher position than the height of the gravity center 31 in the engine 2.

On one hand, the engine 2 is supported at the sub-frame 22 which are supported through the ~~[[rear]]~~ front mount 11a disposed at the front side of the engine 2, the rear mount 11b disposed at the back of the engine 2, and the trans-lower mount and by the body side frames 23 through the ~~floating~~ sub-frame support mounts 13.

On the other hand, the engine 2 is supported at the body side frames 23 through the side engine mount 12a and trans-upper mount 12b disposed at the right and left sides of the engine 2, respectively.

Here as described above, on one hand, the height of the center of elasticity, center which is composed of the front mount 11a, rear mount 11b, and trans-lower mount, is set at a lower position than the height of the gravity center 31 in the engine 2.

On the other hand, the height of the center of elasticity, center which is composed of the side engine mount 12a and trans-upper ~~mount 12b~~ mount 12b is set at a higher position than the height of the gravity center 31 in the engine 2.

Thus, the height of the center of elasticity center as a whole in the plurality of mount members 1 which are composed of the front mount 11a, rear mount 11b, trans-lower mount, side engine mount 12a, and trans-upper ~~mount 12b~~ mount 12b is intended to be set at a higher position than the height of the gravity center 31 in the engine 2.

As shown in FIG. 5A, when a vibratory force (see a black bold arrow mark) acting in the lateral directions through a vehicle suspension

~~suspensions~~ is applied to the supporting structure, ~~[[such]]~~ the plurality of mount members 1 allow for converting the vibration characteristics of the engine 2 into a rotational vibration about a center (see a black small arrow mark in FIG. 5A) which is higher than the height of the gravity center 31. In this way, the characteristics of vibration transmissions to the vehicle body 21 can be adapted so that the transmitted vibration can be reduced in both the direction of the vibratory force through the ~~suspensions~~ suspension and a vehicle travel direction.

On the other hand, in a conventional supporting structure, of which ~~elasticity~~ a center of elasticity as a whole is set to be lower than the height of gravity center of an engine, a vibration in the lateral direction will be rather large, in contrast to ~~different from~~ the supporting structure of the vehicle power source according to the illustrated embodiment (see a white arrow mark in FIG. 5A).

As described above, such the supporting structure results in the elimination of vibration feelings in the longitudinal ~~direction~~ direction as well as in the lateral direction of the vehicle body 21 because the height of the center of elasticity center is designed to consist of the primary weight sharing mount members 11 and secondary weight sharing mount members 12, which are the plurality of the mount members 1, is set at a higher position than the height of the gravity center 31 in the engine 2.

Accordingly, the supporting structure enables a vibration isolation effect and a drive-safety/ride-quality feeling with a sense of ~~oneness of~~ unity between the power source and vehicle body to be surely and sufficiently obtained.

WHAT IS CLAIMED IS:

1. A supporting structure of a vehicle power source comprising:

a plurality of mount members with elasticity through which a power source is mounted on a vehicle body;

first mount members for primarily sharing the weight of the power source and at least one of the first mount members, which is supported by said vehicle body at a first height lower than the height of gravity center of said power source, is attached to a sub-frame; and

second mount members for secondarily sharing the weight of the power source and the second mount members, which have a spring effect in at least one of longitudinal and lateral directions of said vehicle body, are attached to the vehicle body at a second height higher than the gravity center of said power source,

wherein the height of elasticity center of the structure defined by the first and second mount members is set to be higher than the gravity center of said power source.

2. A supporting structure of a vehicle power source according to claim 1, wherein said power source comprises an engine, a crank shaft which is disposed in the lateral direction, and a transmission is connected to one end of the crank shaft, wherein said first members comprise a front mount disposed at a front side of said engine and a rear mount disposed at a back side of the engine, and wherein said second members comprise a side engine mount disposed at an end opposite to said transmission and a trans-upper mount disposed on the transmission.

3. A supporting structure of a vehicle power source according to claim 1, wherein said spring effect of said second members is adapted to be softer in

a vertical direction than one of longitudinal and lateral directions.

4. A supporting structure of a vehicle power source according to claim 1, wherein said sub-frame is supported under body side frames through floating mounts comprising bolts and buffer members.

ABSTRACT OF THE DISCLOSURE

A supporting structure of a vehicle power source consists of a plurality of mount members ~~with~~ having elasticity through which a power source is mounted on a vehicle body; first mount members for primarily sharing the weight of the power source and at least one of the first mount members, which
5 is supported by the vehicle body at a first height lower than the height of a gravity center of the power source, is attached to a sub-frame; and second mount members for secondarily sharing the weight of the power source and the second mount members, which ~~have a spring effect in at least one of~~
10 ~~longitudinal and lateral directions of the vehicle body,~~ are attached to the vehicle body at a second height higher than the gravity center of the power source. ~~In the supporting structure, the~~ A height of a center of elasticity center of the supporting structure defined by the first and second mount members is set to be higher than a height of the gravity center of the power source.